

10<sup>th</sup> International Physics Olympiad  
1977, Hradec Králové, Czechoslovakia

**Problem 1.** The compression ratio of a four-stroke internal combustion engine is  $\varepsilon = 9.5$ . The engine draws in air and gaseous fuel at a temperature  $27^\circ\text{C}$  at a pressure  $1 \text{ atm} = 100 \text{ kPa}$ . Compression follows an adiabatic process from point 1 to point 2, see Fig. 1. The pressure in the cylinder is doubled during the mixture ignition (2–3). The hot exhaust gas expands adiabatically to the volume  $V_2$  pushing the piston downwards (3–4). Then the exhaust valve opens and the pressure gets back to the initial value of  $1 \text{ atm}$ . All processes in the cylinder are supposed to be ideal. The Poisson constant (i.e. the ratio of specific heats  $C_p/C_V$ ) for the mixture and exhaust gas is  $\kappa = 1.40$ . (The compression ratio is the ratio of the volume of the cylinder when the piston is at the bottom to the volume when the piston is at the top.)

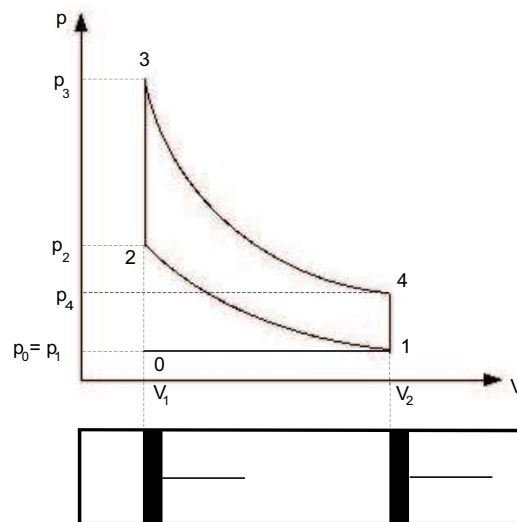


Figure 1:

- a) Which processes run between the points 0–1, 2–3, 4–1, 1–0?
- b) Determine the pressure and the temperature in the states 1, 2, 3 and 4.
- c) Find the thermal efficiency of the cycle.
- d) Discuss obtained results. Are they realistic?

*Solution:* a) The description of the processes between particular points is the following:

0–1 :	intake stroke	isobaric and isothermal process
1–2 :	compression of the mixture	adiabatic process
2–3 :	mixture ignition	isochoric process
3–4 :	expansion of the exhaust gas	adiabatic process
4–1 :	exhaust	isochoric process
1–0 :	exhaust	isobaric process

Let us denote the initial volume of the cylinder before induction at the point 0 by  $V_1$ , after induction at the point 1 by  $V_2$  and the temperatures at the particular points by  $T_0$ ,  $T_1$ ,  $T_2$ ,  $T_3$  and  $T_4$ .

b) The equations for particular processes are as follows.

0–1 : The fuel-air mixture is drawn into the cylinder at the temperature of  $T_0 = T_1 = 300$  K and a pressure of  $p_0 = p_1 = 0.10$  MPa.

1–2 : Since the compression is very fast, one can suppose the process to be adiabatic. Hence:

$$p_1 V_2^\kappa = p_2 V_1^\kappa \quad \text{and} \quad \frac{p_1 V_2}{T_1} = \frac{p_2 V_1}{T_2}.$$

From the first equation one obtains

$$p_2 = p_1 \left( \frac{V_2}{V_1} \right)^\kappa = p_1 \varepsilon^\kappa$$

and by the dividing of both equations we arrive after a straightforward calculation at

$$T_1 V_2^{\kappa-1} = T_2 V_1^{\kappa-1}, \quad T_2 = T_1 \left( \frac{V_2}{V_1} \right)^{\kappa-1} = T_1 \varepsilon^{\kappa-1}.$$

For given values  $\kappa = 1.40$ ,  $\varepsilon = 9.5$ ,  $p_1 = 0.10$  MPa,  $T_1 = 300$  K we have  $p_2 = 2.34$  MPa and  $T_2 = 738$  K ( $t_2 = 465$  °C).